


| Name: | |  | |
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| Enrolment No: | | | |
| UNIVERSITY OF PETROLEUM AND ENERGY STUDIES End Semester Examination, May 2022 | | | |
| Course: Solar Thermal Systems Program: B.Tech Mechanical Course Code: MECH 4028P | | Semester: VIII Time : 03 hrs. Max. Marks: 100 | |
| Instructions: Attempt all questions | | | |
| SECTION A (5Qx4M=20Marks) | | | |
| S. No. | | Marks | CO |
| Q.1 | Draw diagram to Illustrate the angle of incidence, the zenith angle and the solar altitude angle? | 4 | CO1 |
| Q.2 | Explain the variation of temperature along x, y and z axis of liquid flat plate collector? | 4 | CO1 |
| Q.3 | Explain the effect of number of covers on collector performance considering selective surfaces? | 4 | CO2 |
| Q.4 | Explain the limitations of two tank molten salt TES systems? | 4 | CO1 |
| Q.5 | Explain the performance characteristic of solar cell? | 4 | CO1 |
| SECTION B (4Qx10M= 40 Marks) | | | |
| Q.6 | Calculate the angle made by beam radiation with the normal to a flat plate collector on May 1 at 0900 h LAT. The collector is located in New Delhi (28°35'N, 77°12'E). It is tilted at an angle of 36° with the horizontal and is pointing due south. | 10 | CO3 |
| Q.7 | Explain energy balance equation of flat plate collector considering top loss, bottom loss and side loss coefficient. | 10 | CO2 |
| Q.8 | Explain Thermal Ratcheting and geometrical modification required to avoid failure due to thermal ratcheting? | 10 | CO1 |
| Q.9 | Draw layout of concentrated solar power plant with and without thermal energy storage systems | 10 | CO2 |
| SECTION-C (2Qx20M=40 Marks) | | | |
| Q.10 | A cylindrical parabolic focusing collector is used for heating a thermic fluid ($C_p = 2.2 \text{ kJ/kg-K}$) which enters with a temperature of 160°C . The concentrator has an aperture of 1.8 m and a length of 3 m. The absorber tube has an inner diameter of 2.8 cm and outer diameter of 3.2 cm and has a concentric glass cover around it. Given that: Specular reflectivity of concentrator surface : 0.82 Intercept factor : 0.91 $(\tau\alpha)_b : 0.8$ | 20 | CO3 |

| | <p>Beam radiation incident normally on aperture plane : $152 \text{ W/m}^2\text{-K}$ Overall loss coefficient : $9.5 \text{ W/m}^2\text{-K}$ Convective heat transfer coefficient on inside of absorber tube : $325 \text{ W/m}^2\text{-K}$ Ambient Temperature : $27 \text{ }^\circ\text{C}$ Mass flow rate of the fluid : 360 kg/h Calculate the useful heat gain rate, the exit temperature of the fluid and the instantaneous efficiency.</p> | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
|---|--|--------------------------|------------|--------------------------|-----|---------------------|---|----------|---|---|---|---|----------|---|---|---|---|----------|---|---|---|---|-----|----|---|----|---|----|----|---|---|---|-----------------------|------|---|----|---|---------------------------------|------|---|----|----|---------|----|---|---|----|----------------|---|---|---|---------------------|-------|--------------------|-------|--------------------------|-----|-------------------------|-----|---|--------|---------------|----------|------------------------------------|-----------------------------|---------------------|-----------------------------|-------|---------------------|----------------------|------|--------------------------------|------------------------------|-----------------------------------|------------------------------|---------------------------|------|------------------|-------------------|
| <p>Q.11</p> | <p>Evaluate annual energy usage of a residential house based on the appliances and power consumption given in table below. Also calculate the number of PV panel require if each PV panel wattage capacity is 220 watt</p> <table border="1" data-bbox="240 663 1149 1167"> <thead> <tr> <th>S.No</th> <th>Appliances</th> <th>Power consumption (watt)</th> <th>Nos</th> <th>Hours used each day</th> </tr> </thead> <tbody> <tr> <td>1</td> <td>LED Bulb</td> <td>9</td> <td>3</td> <td>6</td> </tr> <tr> <td>2</td> <td>LED Bulb</td> <td>9</td> <td>3</td> <td>8</td> </tr> <tr> <td>3</td> <td>LED Bulb</td> <td>9</td> <td>4</td> <td>6</td> </tr> <tr> <td>4</td> <td>Fan</td> <td>75</td> <td>3</td> <td>16</td> </tr> <tr> <td>5</td> <td>TV</td> <td>40</td> <td>1</td> <td>5</td> </tr> <tr> <td>7</td> <td>AC (1.5 ton - 5 Star)</td> <td>1350</td> <td>1</td> <td>10</td> </tr> <tr> <td>9</td> <td>Fridge (@ ENERGY STAR - 5 Star)</td> <td>24.5</td> <td>1</td> <td>24</td> </tr> <tr> <td>11</td> <td>Speaker</td> <td>25</td> <td>1</td> <td>5</td> </tr> <tr> <td>12</td> <td>Mobile charger</td> <td>4</td> <td>1</td> <td>6</td> </tr> </tbody> </table> <p style="text-align: center;">OR</p> <p>Calculate the performance of a conventional solar air heater with the following data:</p> <table data-bbox="253 1335 873 1841"> <tr> <td>Length of collector</td> <td>2.1 m</td> </tr> <tr> <td>Width of collector</td> <td>1.1 m</td> </tr> <tr> <td>Length of absorber plate</td> <td>2 m</td> </tr> <tr> <td>Width of absorber plate</td> <td>1 m</td> </tr> <tr> <td>Spacing between absorber plate and bottom plate</td> <td>1.5 cm</td> </tr> <tr> <td>Air flow rate</td> <td>200 kg/h</td> </tr> <tr> <td>Air inlet temperature (T_{fi})</td> <td>$50 \text{ }^\circ\text{C}$</td> </tr> <tr> <td>Ambient temperature</td> <td>$20 \text{ }^\circ\text{C}$</td> </tr> <tr> <td>$I_T$</td> <td>$950 \text{ W/m}^2$</td> </tr> <tr> <td>$(\tau\alpha)_{avg}$</td> <td>0.85</td> </tr> <tr> <td>Top loss coefficient (U_t)</td> <td>$6.2 \text{ W/m}^2\text{-K}$</td> </tr> <tr> <td>Bottom loss coefficient (U_b)</td> <td>$0.8 \text{ W/m}^2\text{-K}$</td> </tr> <tr> <td>$\epsilon_b = \epsilon_p$</td> <td>0.95</td> </tr> </table> | S.No | Appliances | Power consumption (watt) | Nos | Hours used each day | 1 | LED Bulb | 9 | 3 | 6 | 2 | LED Bulb | 9 | 3 | 8 | 3 | LED Bulb | 9 | 4 | 6 | 4 | Fan | 75 | 3 | 16 | 5 | TV | 40 | 1 | 5 | 7 | AC (1.5 ton - 5 Star) | 1350 | 1 | 10 | 9 | Fridge (@ ENERGY STAR - 5 Star) | 24.5 | 1 | 24 | 11 | Speaker | 25 | 1 | 5 | 12 | Mobile charger | 4 | 1 | 6 | Length of collector | 2.1 m | Width of collector | 1.1 m | Length of absorber plate | 2 m | Width of absorber plate | 1 m | Spacing between absorber plate and bottom plate | 1.5 cm | Air flow rate | 200 kg/h | Air inlet temperature (T_{fi}) | $50 \text{ }^\circ\text{C}$ | Ambient temperature | $20 \text{ }^\circ\text{C}$ | I_T | 950 W/m^2 | $(\tau\alpha)_{avg}$ | 0.85 | Top loss coefficient (U_t) | $6.2 \text{ W/m}^2\text{-K}$ | Bottom loss coefficient (U_b) | $0.8 \text{ W/m}^2\text{-K}$ | $\epsilon_b = \epsilon_p$ | 0.95 | <p>20</p> | <p>CO3</p> |
| S.No | Appliances | Power consumption (watt) | Nos | Hours used each day | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 1 | LED Bulb | 9 | 3 | 6 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 2 | LED Bulb | 9 | 3 | 8 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 3 | LED Bulb | 9 | 4 | 6 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 4 | Fan | 75 | 3 | 16 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 5 | TV | 40 | 1 | 5 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 7 | AC (1.5 ton - 5 Star) | 1350 | 1 | 10 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 9 | Fridge (@ ENERGY STAR - 5 Star) | 24.5 | 1 | 24 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 11 | Speaker | 25 | 1 | 5 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| 12 | Mobile charger | 4 | 1 | 6 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Length of collector | 2.1 m | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Width of collector | 1.1 m | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Length of absorber plate | 2 m | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Width of absorber plate | 1 m | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Spacing between absorber plate and bottom plate | 1.5 cm | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Air flow rate | 200 kg/h | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Air inlet temperature (T_{fi}) | $50 \text{ }^\circ\text{C}$ | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Ambient temperature | $20 \text{ }^\circ\text{C}$ | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| I_T | 950 W/m^2 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| $(\tau\alpha)_{avg}$ | 0.85 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Top loss coefficient (U_t) | $6.2 \text{ W/m}^2\text{-K}$ | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| Bottom loss coefficient (U_b) | $0.8 \text{ W/m}^2\text{-K}$ | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |
| $\epsilon_b = \epsilon_p$ | 0.95 | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | | |

Useful Empirical Relations

1. Declination Angle, δ (in degree) = $23.45 \sin \left[\frac{360}{365} (284 + n) \right]$ where, n = no. of days
2. Incident angle, $\cos \theta = \sin \varphi (\sin \delta \cos \beta + \cos \delta \cos \gamma \cos \omega \sin \beta) + \cos \varphi (\cos \delta \cos \omega \cos \beta - \sin \delta \cos \gamma \sin \beta) + \cos \delta \sin \gamma \sin \omega \sin \beta$
3. Liquid Flat plate collector-Collector efficiency factor: $F' = \frac{1}{WU_l \left[\frac{1}{U_l[(W-D_o)\varphi + D_o]} + \frac{\delta_A}{k_A d_o} + \frac{1}{3.14 D_i h_f} \right]}$
4. Liquid Flat plate collector – Heat removal factor: $F_R = \frac{\dot{m}C_p}{U_l A_p} \left[1 - \exp \left\{ -\frac{F' U_l A_p}{\dot{m}C_p} \right\} \right]$
5. Parabolic collector-Collector Efficiency factor: $F' = \frac{1}{U_l \left[\frac{1}{U_l} + \frac{D_o}{D_i h_f} \right]}$
6. Parabolic Trough Collector – Heat Removal Factor : $F_R = \frac{\dot{m}C_p}{3.14 * D_o U_l L} \left[1 - \exp \left\{ -\frac{F' p i * U_l D_o L}{\dot{m}C_p} \right\} \right]$